

# Investigation of Influence of Injector Opening Pressure and Nozzle Geometry on the Performance and Emission Characteristics of DI Diesel Engine with CAOME

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# ABSTRACT

The diminution of fossil fuels and stringent pollution norms obliging researchers to search for fuels that result in less pollution and the one that suits to power compressed ignition engines. In the current work the effect IOP (Injector opening pressure) and nozzle geometry on the performance and emission characteristics of diesel engine fueled by Castor oil methyl esters (CAOME) is reported. For a 5-hole nozzle at 240 bar IOP the highest brake power is obtained (27.32%). Peak pressure gradual upturn was observed when the IOP was varied from 210 bar to 240 bar and the peak value of pressure is 79 bar at IOP of 240 bar.

Keywords: inject opening pressure, injection timing, castor oil methyl esters

# INTRODUCTION

The progress of any country mainly be influenced by on the available energy resources. With ever increase in energy consumption, rigorous pollution norms and depletion of fossil fuels resulted in need for huge investment in energy sector to fulfill the requirement and to investigate new sources of ecofriendly energy resources. Energy is the vital factor that decides the socio-economic growth of human society. Mechanization and modernization resulted in abrupt rise in energy demand and henceforth the exploration of the same which are mainly fossil fuel based. This situation resulted in search for new alternatives and one among them is oils obtained from green matter. Oils obtained from sunflower, soybean, cottonseed, canola, jatropha, corn, peanut oil etc. are in comparison with petroleum based fuels. These oils can be used in diesel engines with several modifications such as preheating to decrease viscosity, Varying IOP and injection timing.

#### **First-generation Biofuels**

Ethanol synthesized by fermenting sugar extracted from sugar, sugar beets, sugar extracts from starch that present in maize kernels and other feedstock are considered as first generation biofuel. The process utilizes different organisms that can yields alcohol and butanol. Fuel ethanol production in the year 2017 was led by United States with 15800 million gallons followed by Brazil with 7060 million gallons and European Union with 1415

million gallons. The trend of bio ethanol production in India is in uptrend and has reached 280 million gallons in the year 2017.

#### **Second-generation Biofuels**

Biofuels that are considered under Second generation are synthesized from ligno cellulosic biomass, facilitating the use of low cost, non-edible feed stocks which limits the conclusion between food and fuel demand. Further the second generation biofuels are classified based on the process used for the conversion of green matter into fuel that may be bio or thermo chemical. Second generation ethanol or butanol are produced by biochemical process whereas thermo chemical process is used for generation of other second generation fuels. The fuels produced by thermo chemical process are not much familiar even they found commercial success which includes dimethyl ether, methanol and Fischer-Tropsch liquids. As we know biomass has found its application as animal fodder, as raw material in fertilizer industry and in paper making. Mammoth magnitudes of farming trashes are generated annually worldwide. Even though biomass has great potential in terms of energy content, large quantities are disposed that has ill effects on environment.

As the energy consumption is increasing and fossil fuels are depleting, the research related to alternative sources to fuel internal combustion (IC) engines has got great potential. Especially the biofuels produced from green matter maintains carbon balance in nature and may replace conventional fuels in the near future. Castor oil is a potential source to produce biodiesel and is suitable to use in IC engines (Arunkumar et al., 2019). A detailed study of the literature relevant to the effect of injection timing (IT) and injection operating (IOP) was also studied.

Panwar et al. (2010), have evaluated the engine Performance when fueled with different blends of CAOME. In this work the biofuel was prepared by transesterification using potassium hydroxide (KOH) as catalyst and properties of CAOME are in comparison with diesel. The engine performance parameters are analyzed with different blends of biodiesel and are in comparison with diesel. Only 2% difference in calorific value was found between diesel and B10 blend (45.50 MJ/kg). Nitrogen oxide (NOx) emissions for CAOME are similar trend as of diesel at lower loads and found deviation at full loads. Hence CAOME can be used as an alternative fuel in diesel engines. Barajas Forero (2005), has studied the production of castor oil biodiesel by transesterification using NaOH catalyst and the results prove that it better suits for cold weather operation. Properties of biofuel were tested according to American Society for Testing and Materials (ASTM) D6751 standard and are in comparison with petroleum diesel and B100 being exception due to high viscosity and humidity. The viscosity of mixture increased with increase in proportion of biodiesel but not affected the atomization characteristics.

Imankular (2012), has studied the biodiesel synthesis technique from castor oil and evaluated physical and chemical characteristics of oil including infrared (IR) spectrum. The properties of biodiesel are in comparison with diesel. The value of coking capacity of 10% the residue was within the limit of norms. The results shows that optimum concentration of addition of the bio additive equal to 5% of mass. IR spectrum of biodiesel indicates the fluctuation of hydroxyl group of ricinoleic acid which also allows distinguishing it from other fatty acids and castor acid itself forms other natural substances. Okullo et al. (2012) have evaluated physico-chemical properties of biodiesel prepared from jatropha oil and castor oil. In this study castor oil biodiesel was produced in the presence of methanol and sodium hydroxide (NaOH) catalyst. The evaluation of physico- chemical properties of castor oil are significant because of the presence of exceptionally high ricinoleic acid. The low temperature suitability (Barajas Forero, 2005) and its ability for complete solubility in alcohol (Imankulov, 2012) makes it suitable for biodiesel production. Study reveals that biodiesel derived from jatropha oil suits its usage in diesel engine whereas biodiesel produced from castor oil (B100) is not suitable due to high viscosity but the same can be reduced by refining castor oil prior to transesterification. The study also reveals that neutralization of oils generally improves the quality of biodiesel by lowering the amount of free fatty acids and thus increasing yield. Ganeshan and Elango (2013) have investigated the performance of compression ignition engine for different blends of castor oil biodiesel and ethanol. The experiment was conducted on single cylinder Compression Ignition engine, the result obtained for a blend with 70% diesel, 10% castor oil and 20% ethanol gives optimum values of performance and also shown decrease in NOx and unburnt hydrocarbon emissions. In this study results obtained by experiment are compared with the one obtained computational fluid dynamics and are found to be in great accuracy. Wilson and Yalini (2015) have investigated the performance and emission characteristics of CI engine using multiple blends of castor oil in single cylinder four stroke compression ignition engine at different load conditions with different shapes of piston. In this study the castor oil biofuel is produced in two stage process, first being acid catalyzed esterification to reduce free fatty acid content below 1%, the next is base catalyzed transesterification to convert oil to biodiesel and resulted in 90% yield rate. Experiment have been conducted with different proportions based on volume basis B100, B15, B35 and B45, the results are compared with the one obtained for diesel. The investigation reported that methyl ester of castor oil gives better performance when blended with diesel for toroidal shapes of piston. It is also observed that for the same piston shape specific fuel consumption and power obtained is maximum. Deshpande et al. (2012) have studied the method of production of castor oil biodiesel use acid and base catalyst.

In this work the base and acid catalyst used are sodium hydroxide (NaOH) and Sulfuric acid (H<sub>2</sub>SO<sub>4</sub>), the influence of process parameters such as temperature, residue time, and catalyst concentration on yield are also presented. The results shows that with sulfuric acid as catalyst with increase in reaction time from 30 minutes to 45 minutes resulted in decrease in viscosity from 16.56 cSt to 11.28cSt and further it observed that there is an increase in viscosity with increase in reaction time from 30 minutes to 45 minutes as catalyst, with increase in reaction time from 30 minutes to 45 minutes results in decrease in viscosity from 21.57 cSt to 13.10 cSt and optimum reaction temperature was found to be 30°C which results in castor oil with lowest viscosity. Sreenivas et al. (2011) have studied the process of production of castor oil methyl esters in the presence of methanol and NaOH catalyst. The properties of produced biofuel were evaluated and found to be comparison with diesel as per ASTM and Indian biofuel standards. The fuel is biodegradable, non-toxic, high flash point and hence safe for transportation and storage. Higher cetane number is achieved hence increases combustion efficiency and decreases emissions.

#### Injector opening pressure

Injector opening pressure and injection timing significantly influence the performance and emission characteristics of IC engines and in past it was reported by many researchers, some are discussed here. Dharmadhikari et al. (2012) have evaluated the performance and emission characteristics of compressed ignition engine using blends of biodiesel and diesel at different injection pressure. In the present work neem oil methyl ester (NOME) and karanja oil methyl ester (KOME) are blended with diesel. The experiments are conducted for injection pressure (IP) range from 180 to 220 bar and 200 bar injection pressure was found to be the optimum value. For B10 and B20 blend the HC, CO emissions are found to be less compared to that of diesel. The brake thermal for all blends fall in the range of 6% less in comparison with diesel. Blends B10 and B20 exhibits higher values of brake specific fuel consumption at higher loads compared to that of diesel. Jindal et al. (2010) have investigated the influence of compression ratio (16, 17 and 18) and injection pressure (150, 200 and 250 bar) in direct injection (DI) diesel engine fueled with Jatropha methyl esters. The results shows the trend of decreasing brake specific fuel consumption with increase in brake thermal efficiency and increases with combined increase in compression ratio and injection pressure. Hydrocarbon, NOx, smoke opacity and exhaust gas temperature is found lower for all combinations of compression ratio and injection pressure in comparison with diesel with same operating parameters. The optimum combination of compression ratio and injection pressure was found to be 18 and 250 bar respectively. Balasubbramanian et al. (2009) have studied the effect of IOP (200, 220 and 240 bar) on performance, emission and combustion characteristics of DI diesel engine fueled with linseed oil methyl esters. It was found that optimum injection pressure was 240 bar which gives thermal efficiency almost equal to diesel with decreased carbon monoxide and unburnt hydrocarbon emissions but there is a slight increase in NOx emissions. Nwafor et al. (2000) have investigated the effect on performance of diesel engine fueled with rapeseed oil with advanced injection timing. In diesel cycle the fuel is injected some degrees before top dead centre into the hot compressed cylinder gas because combustion process initiated by self-ignition. Combustion delay and lag in burning is observed in engines that uses alternate fuels hence the advancement of the injection timing will compensate for the same. The optimum injection timing results in maximum power and occurred within 5° crank angle after top dead centre, hence injection timing of fuel was 3.5° advanced and the inconsistent behavior of the engine was perceived with further rise in the same (Panneerselvam et al., 2015).

From the literature review undertaken on the production of biofuel, effect of operating parameters and the factors influencing the performance and emissions, it is observed that the research on the methods to reduce the viscosity has got great potential. Different injector opening pressure (IOP), the combustion efficiency analysis by using variable nozzle geometry and different combustion chamber shapes has great influence on the performance and emission characteristics of engine hence research related to that has got great prominence. The objective of the present work is to investigate the influence of IOP and nozzle geometry on the engine output characteristics.

### DESCRIPTION

#### Materials Used

Experiments are conducted on 4 stroke single cylinder engine fitted with hemispherical combustion chamber with varied engine pressure (210 bar to 260 bar) and different nozzle geometry and the performance and emission characteristics are evaluated.

Castor oil was purchased from the local industry and the properties of the same are evaluated and mentioned in **Table 1**.

Table 1. Raw Castor Oil Properties					
Properties	Castor oil Methyl esters				
Flash point	320°C	320°C			
Fire point	345°C	345°C			
Kinematic viscosity	52cSt.	52cSt.			
Density	956 kg/m <sup>3</sup>				
F <sub>1</sub> Fuel flow rate sensor	T <sub>3</sub> Temperature of the calorimeter water inlet				
F <sub>2</sub> Air flow rate measuring sensor	T <sub>4</sub> Temperature of calorimeter outlet				
F <sub>3</sub> Flow rate to the jacket water measurement	t T <sub>5</sub> Temperature of exhaust gas before calorimeter				
F <sub>4</sub> Water flow inlet to the calorimeter	T <sub>6</sub> Temperature of exhaust gas after calorimeter				
T <sub>1</sub> Water jacket inlet temperature	PT Pressure sensor at combustion chamber				
T <sub>2</sub> Water jacket outlet temperature	Wt Weight				
	Air PT T2 F3 F4 Water Water inlet ENGINE				

Figure 1. Schematic line diagram of engine test setup

#### Table 2. Properties of Biodiesel

Diesel (%)	Biodiesel (%) CAOME	Density (kg/m <sup>3</sup> )	Viscosity (cSt)	Calorific Value (CV) (kJ/kg)	Specific gravity	Flash point (°C)
100	-	834	2.38	42250	0.834	60
-	100	927	5.57	37730	0.927	189

### Table 3. Engine specifications of tested engine

Parameters	Specifications		
Engine type	TV1 (Kirloskar make)		
Software used	Engine soft		
Injector operating pressure	200 to 225 bar		
Static injection time	23°BTDC		
Governor type	Centrifugal type Mechanical		
No of cylinders	Single cylinder		
No of strokes	4 stroke		
Fuel oil	High Speed Diesel		
Rated power	5.2 kW at 1500 rpm		
Cylinder diameter (Bore)	0.0875 m		
Stroke length	0.11 m		
Ratio of compression	17.5:1		

The biodiesel is obtained from raw castor oil by transesterification, the process of removing fatty acid content in the presence of catalyst to produce the ethyl ester or methyl esters. The properties of castor oil biofuel are evaluated as per ASTM standards (Subcommittee: D02.E0 2009) are mentioned in Table 2.



Figure 2. Brake Thermal Efficiency vs Brake power at different IOP

# EXPERIMENTAL RESULTS AND DISCUSSION

#### Optimization of Injector Opening Pressure (IOP) for Fuels Tested

The research has been carried out to identify the optimum injector opening pressure and nozzle geometry on the biodiesel fueled modified diesel engine.

In the present work, studies on basic performance and emission characteristics of the engine were carried out on the normal diesel engine using CAOME at different injection pressures. The injector opening pressures were varied from 210 bar to 260 bar, for differential loading when the engine was running at 1500 rpm. Fuel and air flow rate, temperature of exhaust gas, Smoke, hydrocarbon (HC), nitrogen oxides (NOx) and carbon monoxide (CO) emissions were chronicled. The optimum injection pressure is identified based on results and the same is fixed in the further work to evaluate engine performance, emissions and combustion parameters. For diesel mode the engine is operated as per manufacturer specification IOP of 205 bar. For 3, 4 and 5-hole (0.3mm dia) nozzle and at an injection timing of 27°BTDC the effect of IOP in the range of 210 bar to 260 bar is presented in the following graphs.

### Effect of Nozzle Geometry and Injector Opening Pressure

#### Brake thermal efficiency

For hemispherical combustion chamber at different IOP the effect of break power is shown in **Figure 2** for different nozzle geometry. 240 bar injection pressure is found optimum as it resulted in highest brake thermal efficiency due to improved atomization, better spray characteristics and better mixing of fuel with air and which in turn improved combustion (Agarwal et al., 2014). Further rise in IOP (260 bar) results in injection delay and which in turn negates the gain of higher IOP's (Raeie et al., 2014). From the experimental results it is clearly observed that at 240 bar IOP higher thermal efficiency (27.32%) is produced when the engine is fitted with 5-hole nozzle in comparison with 3 hole and 4-hole nozzles which is due to improved air-fuel mixing rate.

# Smoke opacity

The smoke opacity for nozzle with 3, 4- and 5-hole nozzle for IOP ranging from 200 to 240 bar for brake power output is shown in Figure 3. The increase in IOP decreases the smoke level due to increased spray atomization which is a result of better air fuel mixture formation up to 240 bar and further increase in IOP (260bar) resulted in delayed injection as result the smoke opacity upsurge is reported. It is also observed form Figure 3 that



Figure 3. Smoke opacity v/s injector opening pressure for different nozzle geometry

the increase in number of holes in nozzle reduced the smoke emissions due to standardized air fuel mixture formation.

# HC emission

It is perceived from **Figure 4** that there is a substantial drop in HC emissions as a result of better combustion at 240 bar IOP. In general biofuels are higher viscous in comparison with diesel which will result in ignition delay but the higher IOP's compensate the ignition delay as a result of enhanced atomization (Das et al., 2018). Further increase in IOP (260 bar) results in injection delay quashing the gain of higher IOP's. Better results are reported at 240 bar IOP for a 5 hole nozzle.



Figure 4. Unburned Hydrocarbon Emissions v/s injector opening pressure



Figure 5. Carbon Monoxide Emissions v/s injector opening pressure

# CO emission

Better combustion occurs at 240 bar IOP resulted in lower CO emissions which follows the similar trend as of HC emissions. Best results in terms of emissions are reported at 240 bar IOP for a 5 hole nozzle.

# NOx emission for different nozzle geometry

Higher engine cylinder temperatures are reported due to enhanced combustion efficiency as result the upturn in nitric oxide emissions is observed, the peak values of the same are reported at 240 bar IOP for a 5-hole nozzle (Bueno et al., 2017).



Figure 6. NO<sub>x</sub> Emissions v/s injector opening pressure



Figure 7. Peak Pressure v/s Brake power at different injector opening pressure

# Peak pressure with brake power

The energy content and viscosity of CAOME is less in comparison with diesel hence lower peak pressure is observed due to ignition delay, slow burning rate, lower adiabatic flame temperature as a result of incomplete combustion due to improper mixing of fuel and air. Figure 7 shows substantial upturn in peak pressure with increase in IOP from 210 bar to 240 bar and further rise in IOP is adversely affected due to injection delay.

# Ignition delay period with brake power

For a 5-hole nozzle the influence of Injector opening pressure on ignition delay with brake power for diesel and CAOME is illustrated in **Figure 8**. Static injection timing is taken as reference to calculate ignition delay, and it decreased with load and increased with biodiesel operation and the same is observed when the engine fuelled with CAOME. The delay in ignition can be compensated by increasing IOP which result in better combustion.



Figure 8. Ignition Delay Period v/s Brake Power for different injector opening pressure



Figure 9. Combustion Duration v/s Brake Power for different injector opening pressure

#### Combustion duration with brake power

In the present context the Combustion duration is the time duration between combustion initiation and the time to release 90 percent heat content. It is observed from **Figure 9** that there is a notable rise in combustion duration for both CAOME and diesel with increase in brake power and for IOP as well. CAOME combustion duration trends are higher in comparison with diesel due its viscous nature and less energy content and the same is up to certain extent is compensated by high IOP's.

#### Limitations of the Work

In the present work experiments are conducted on engine fitted with hemispherical combustion chamber. The experiments can be conducted by varying injection timing in engine fitted with different combustion chambers such as toroidal and reverse toroidal other than manufacturer specifications and influence of the same can be evaluated.

#### Conclusion

For a hemispherical combustion chamber, adopting 240 bar injection pressure for a five-hole nozzle is found optimum as it resulted in highest brake thermal efficiency (27.32%) due to improved atomization, better spray characteristics and better mixing of fuel with air and which in turn improved combustion. Gradual decrease in Smoke level, unburnt HC and CO emissions are observed with increased in IOP from 210 to 240 bar. Further rise in IOP (260 bar) results in injection delay and which in turn negates the gain of higher IOP's. The peak value of pressure is 79 bar at IOP of 240 bar and there is a substantial upturn in peak pressure with increase in IOP from 210 bar to 240 bar and further rise in IOP is adversely affected due to injection delay.

Among all IOP and nozzles tested, the best results in terms higher power and less emissions are found for 5hole nozzle and at 240 bar IOP and the same is observed with increase in number of holes in the nozzle.

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